

Notes on
1.63 Advanced Environmental Fluid Mechanics
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December 1, 2002
 3-3Hi-Re-bl.tex

3.3 Viscous Flow at High Reynolds Numbers

Let us first give a heuristic estimates of boundary layer in steady flows.

Consider a particle near the wall to be influenced by viscosity. After traveling a distance x from the edge, it has been under viscous influence for a time of $t = x/U$. Let U be large. For finite x , t is small so that vorticity is spread sideways to the width $(\nu t)^{1/2} \sim (\nu x/U)^{1/2}$. Let us define this width to be the boundary layer, which has thickness $\delta = O(\nu x/U)^{1/2}$.

Alternatively we start from Navier-Stokes equations :

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0 \quad (3.3.1)$$

$$u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = -\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right) \quad (3.3.2)$$

$$u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} = -\frac{1}{\rho} \frac{\partial p}{\partial y} + \nu \left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right) \quad (3.3.3)$$

When viscosity is important $y = O(\delta)$, $x = O(L)$, convective inertia is comparable to viscous stresses.

From continuity

$$\frac{u}{L} \sim \frac{v}{\delta}$$

From x -momentum

$$\begin{aligned} u u_x &\sim \nu u_{yy} \\ U^2 &\sim \nu \frac{U}{\delta^2} \\ \frac{U}{L} &\sim \nu \frac{U}{\delta^2} \end{aligned}$$

Therefore,

$$\delta \sim (\nu L/U)^{1/2} \quad (3.3.4)$$

and

$$\frac{\delta}{L} \sim \left(\frac{\nu}{UL} \right)^{1/2} = Re^{1/2}. \quad (3.3.5)$$

Shear stress on the awall :

$$\frac{\tau_0}{\rho} = \nu \left. \frac{\partial u}{\partial y} \right|_0 = \nu \frac{U}{\delta} \sim \nu U \sqrt{\frac{\bar{U}}{\nu L}}$$

Hence the drag coefficient is,

$$C_D = \frac{\tau_0}{\frac{1}{2}\rho U^2} = 2 \sqrt{\frac{\bar{\nu}}{Ux}} = \frac{2}{\text{Re}}.$$

For water $\nu = 10^{-5} \text{ ft}^2/\text{sec}$. Let $U = 1 \text{ ft}/\text{sec}$ $L = 1 \text{ ft}$, then $\text{Re} = 10^5$. Hence,

$$O\left(\frac{\delta}{L}\right) \propto \frac{1}{\sqrt{\text{Re}}} \sim \frac{1}{3} 10^{-2} \quad (\delta \sim 0.003 \text{ ft})$$

and

$$C_D \sim 0.003.$$

Experiments for flat plates (Schlichting, p. 133) show that: $C_D \sim 0.002$, but experiments for a circular cylinder show that $C_D \approx 0(1)$ because flow is separated for most Re .

3.3.1 Systematic Boundary-layer Approximation

Let $u = O(U)$, $x = O(L)$, $y = O(\delta)$. From continuity, $v = O(U\delta/L)$. Let $u \rightarrow Uu$, $v \rightarrow \frac{U\delta}{L}v$, $x \rightarrow Lx$, $y \rightarrow \delta y$

$$\frac{U}{L} (u_x + v_y) = 0. \quad (3.3.6)$$

$$\frac{U^2}{L} (uu_x + vv_y) = -\frac{P}{\rho L} \frac{\partial p}{\partial x} + \frac{\nu U}{L^2} u_{xx} + \frac{\nu U}{\delta^2} u_{yy}. \quad (3.3.7)$$

$$\frac{\delta}{L} \frac{U^2}{L} (uv_x + vv_y) = -\frac{P}{\rho \delta} p_y + \frac{\nu U}{L^2} \frac{\delta}{L} v_{xx} + \frac{\nu U}{\delta^2} \frac{\delta}{L} v_{yy}. \quad (3.3.8)$$

From Eqn. (3.3.6)

$$u_x + v_y = 0. \quad (3.3.9)$$

From Eqn. (3.3.7)

$$uu_x + vv_y = -\frac{P}{\rho U^2} p_x + \frac{1}{\text{Re}} \left(u_{xx} + \frac{L^2}{\delta^2} u_{yy} \right). \quad (3.3.10)$$

From Eqn. (3.3.8)

$$uv_x + vv_y = -\frac{PL^2}{\rho \delta^2 U^2} p_y + \frac{1}{\text{Re}} \left(v_{xx} + \frac{L^2}{\delta^2} v_{yy} \right). \quad (3.3.11)$$

To keep the dominant viscous stress term in Eqn. (3.3.10), we must have

$$\left(\frac{\delta}{L}\right)^2 = \frac{1}{\text{Re}} \quad \text{or} \quad \frac{\delta}{L} = \text{Re}^{-1/2}. \quad (3.3.12)$$

From Eqn. (3.3.11)

$$p_y = O\left(\frac{\delta^2}{L^2}\right) \quad (3.3.13)$$

and from Eqn. (3.3.10)

$$uu_x + vv_y = -\frac{P}{\rho U^2} p_x + u_{yy}. \quad (3.3.14)$$

In physical variables, we have to leading order

$$u_x + v_y = 0 \quad (3.3.15)$$

$$uu_x + vv_y = -\frac{1}{\rho} p_x + \nu u_{yy} \quad (3.3.16)$$

The pressure is constant across the boundary layer and must be the same as the pressure just outside. In the inviscid outer flow

$$UU_x + VU_y = -\frac{1}{\rho} p_x. \quad (3.3.17)$$

Since $V = 0$ on the wall, $p_x = -\rho UU_x$. Hence, inside the boundary layer:

$$uu_x + vv_y = UU_x + \nu u_{yy}. \quad (3.3.18)$$

This is the classical boundary layer approximation for high Re flows, due to Prandtl (1905).